

1-11. (CANCELED)

12. (NEW) A shiftable multispeed reverse transmission comprising one input shaft (1), a first, a second, a third, a fourth, a fifth, a sixth countershaft (2, 3, 4, 5, 6, 7), one or more gears and idle gears being mounted on the first, the second, the third, the fourth, the fifth, the sixth countershafts (2, 3, 4, 5, 6, 7), the one or more gears and idle gears being non-rotatably connected with the countershafts via shiftable clutches for one of gear or direction change, and one output shaft (8).

the reverse transmission has four forward gears and four reverse gears, a first fixed gear (116) and a first idle gear (115) connected with the input shaft (1) via a first forward clutch (KV) for the four forward gears, the first fixed gear (116) on the input shaft (1) being in constant mesh with a second fixed gear (202) on the first countershaft (2) upon which a reverse clutch (KR) is situated, a second idle gear (102) is connectable via the reverse clutch (KR) with the first countershaft (2) and is in constant connection with a third fixed gear (104), the third fixed gear (104) is additionally in constant mesh with the first idle gear (115) situated upon the input shaft (1) and non-rotatably connected with a third countershaft (4) upon which are situated a fourth fixed gear (204) and a third idle gear (111) connectable via a third clutch (K2) for gear change and in constant mesh with a fifth fixed gear (106) situated upon a fifth countershaft (6) and the fourth fixed gear (204) is in constant mesh with a fourth idle gear (113) situated on the fifth countershaft (6) and capable of being connected via a fourth shiftable clutch (K4) with the fifth countershaft (6), the fifth fixed gear (106) situated on the fifth countershaft (6) being in constant mesh with a sixth fixed gear (107) and the sixth fixed gear (107) being non-rotatably connected with a sixth countershaft (7) and a sun gear (S) of a planetary stage (P) and with an output shaft (8) is non-rotatably connected a seventh fixed gear (108) which is in constant mesh with a fifth idle gear (117), the fifth idle gear (117) being non-rotatably connected with a planet carrier of the planetary stage (P), a hollow shaft of the planetary stage (P) being connectable either via one of a brake (B) with a housing (G) or via a fifth clutch (K) with the sun gear (S).

13. (NEW) The reverse transmission according to claim 12, wherein a first forward gear is engaged by engaging the first forward clutch (KV), the third clutch (K2)

and the brake (B), a second forward gear is engaged by engaging the first forward clutch (KV), the fourth shiftable clutch (K4) and brake (B), a third forward gear is engaged by engaging the first forward clutch (KV), the third clutch (K2) and the fifth clutch (K) of the planetary stage (P), a fourth forward gear is engaged by engaging the first forward clutch (KV), the fourth shiftable clutch (K4) and the fifth clutch (K) of the planetary stage (P).

14. (NEW) The reverse transmission according to claim 12, wherein a first reverse gear is engaged by engaging the reverse clutch (KR), the third clutch (K2) and the brake (B), a second reverse gear is engaged by engaging the reverse clutch (KR), the fourth shiftable clutch (K4) and the brake (B), a third reverse gear is engaged by engaging the reverse clutch (KR), the third clutch (K2) and the fifth clutch (K) of the planetary stage (P), a fourth reverse gear is engaged by engaging the reverse clutch (KR), the fourth shiftable clutch (K4) and of the fifth clutch (K) of the planetary stage (P).

15. (NEW) The reverse transmission according to claim 12, wherein a uniform ratio range is obtained by translation of the planetary stage (P) by shifting a second gear to a third gear.

16. (NEW) A shiftable multispeed reverse transmission comprising one input shaft (1), a first, a second, a third, a fourth, a fifth, a sixth countershaft (2, 3, 4, 5, 6, 7), one or more gears and idle gears being mounted on the first, the second, the third, the fourth, the fifth, the sixth countershafts (2, 3, 4, 5, 6, 7), the one or more gears and idle gears being non-rotatably connected with the countershafts via shiftable clutches for one of gear or direction change, and one output shaft (8),

the reverse transmission has eight forward gears and eight reverse gears, a first idle gear (115), situated upon the input shaft (1) is connected with the input shaft (1) via a forward clutch (SV) which is one of a synchronizer unit or dog clutch for the eight forward gears, the first idle gear (115) being in constant mesh with a first fixed gear (103) which is situated on the second countershaft (3) upon which a second idle gear (110) and a first shift clutch (K1) by way of which the second idle gear (110) can be connected with the first countershaft (3) and a second fixed gear (203) non-rotatably connected with the first countershaft (3) are situated, and upon the input shaft (1) is

situated a third idle gear (116) connectable with the input shaft (1) via a reverse clutch (SR) in the form of one of the synchronizer unit or the dog clutch for the eight reverse gears, the third idle gear (116) being in constant mesh with a third fixed gear (202) upon a first shaft (2) upon which still a fourth fixed gear (102) is situated, and the fourth fixed gear (102) is in constant mesh with a fifth fixed gear (104) which in addition is in constant mesh with the first idle gear (115) situated on the input shaft (1) and is non-rotatably connected with a third countershaft (4) upon which is situated a sixth fixed gear (204) and a fourth idle gear (111) connectable via a second clutch (K2) for gearshift and in constant mesh with a seventh fixed gear (106) situated on fifth countershaft (6) and connectable via a third shift clutch (K4) with a fifth idle gear (113) in constant mesh with the sixth fixed gear (204) of the third countershaft (4), the seventh fixed gear (106) situated on the fifth countershaft (6) being in constant mesh with an eighth fixed gear (107) situated on a sixth countershaft (7) with a planetary stage (P) and the eighth fixed gear (107) is non-rotatably connected with the sixth countershaft (7) and a sun gear (S) of the planetary stage (P) and with the output shaft (8) is non-rotatably connected a ninth fixed gear (108) which is in constant mesh with a sixth idle gear (117), the sixth idle gear (117) being non-rotatably connected with the planet carrier of the planetary stage (P), a hollow gear of the planetary stage (P) being connectable via one of a brake (B) with a housing (G) or via a fourth clutch (K) with the sun gear (S), the eighth fixed gear (107) situated on the sixth countershaft (7) is additionally being in constant operative connection with a tenth fixed gear (105) situated on a fourth countershaft (5), a seventh idle gear (112) upon the fourth countershaft (5) being connectable via a fifth shift clutch (K3) with the fourth countershaft (5), the tenth fixed gear (105) situated upon the fourth countershaft (5) being in constant mesh with the second idle gear (110) which via the first shift clutch (K1) can be connected with the second countershaft (3) and the second fixed gear (203) of the second countershaft (3) is in constant operative connection with the seventh idle gear (112) of the fourth countershaft (5).

17. (NEW) The reverse transmission according to claim 16, wherein a first forward gear is engaged by engaging the forward clutch (SV), the first shift clutch (K1) upon the second countershaft (3) and the brake (B), a second forward gear is engaged

by engaging the forward clutch (SV), the second clutch (K2) of the third countershaft (4) and the brake (B), a third forward gear is engaged by engaging the forward clutch (SV), the fifth clutch (K3) and the break (B), a fourth forward gear is engaged by engaging the forward clutch (SV), the third clutch (K4) and of the brake (B), a fifth forward gear is engaged by engaging the forward clutch (SV), the first shift clutch (K1) and the fourth clutch (K) of the planetary stage (P), a sixth forward gear is engaged by engaging the forward clutch (SV), the second clutch (K2) and the first clutch (K) of the planetary stage (P), a seventh forward gear is engaged by engaging the forward clutch (SV), the third clutch (K3) and the clutch (K) of the planetary stage (P), an eighth forward gear is engaged by engaging the forward clutch (SV), the third clutch (K4) and the first clutch (K) of the planetary stage (P).

18. (NEW) The reverse transmission according to claim 16, wherein a first reverse gear is engaged by engaging the reverse clutch (SR), the first shift clutch (K1) and the brake (B), a second reverse gear is engaged by engaging the reverse clutch (SR), the second clutch (K2) and the brake (B), a third reverse gear is engaged by engaging the reverse clutch (SR), the fifth clutch (K3) and the brake (B), a fourth reverse gear is engaged by enging the reverse clutch (SR), the third clutch (K4) and the brake (B), a fifth reverse gear is engaged by engaging the reverse clutch (SR), the first shift clutch (K1) and the fourth clutch (K) of the planetary stage (P), a sixth reverse gear is engaged by engaging the reverse clutch (SR), the second clutch (K2) and the fourth clutch (K) of the planetary stage (P), an eighth reverse gear is engaged by engaging the reverse clutch (SR), the third clutch (K4) and the second clutch (K) of the planetary stage (P).

19. (NEW) The reverse transmission according to claim 16, where a ratio of the planetary stage (P) is selected so that by changing of a fourth gear to a fifth gear, a uniform ratio range is obtained.